

**Power System Dynamics, Control and Monitoring**  
**Prof. Debapriya Das**  
**Department of Electrical Engineering**  
**Indian Institute of Technology, Kharagpur**

**Lecture – 12**  
**Power System stability (Contd.)**

So just in that previous equation right previous equation. We just solved that is your  $T_m$  minus  $T_e$  divided by  $V A_{base}$  divided by  $\omega_0$ .

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Noting that  $T_{base} = \frac{VA_{base}}{\omega_0}$ ,  $\frac{T_m - T_e}{\frac{VA_{base}}{\omega_0}}$  (111)

$2H \frac{d\omega_r}{dt} = T_m - T_e$  (178)

Where  $\omega_r = \frac{\omega_m}{\omega_0} = \frac{\omega_p}{\omega_b} = \frac{\omega_p}{\omega_b} \cdot \frac{T_{base}}{T_{base}}$

where  $\omega_r$  is angular velocity of the rotor

And this is basically nothing but a torque base, this is nothing but the torque base, then with this equation will become  $T_m$  by torque base minus your  $T_e$  by torque base right. So, this  $T_f$  by  $T_{base}$  is equal to  $T_{bar}$   $T_m$  bar we are writing here and  $T$  by  $T_{base}$  also  $T_e$  bar right. So, in per unit this is per unit and this one  $\omega_r$  upon  $d t$  we are writing previous from previous equation we have written know  $\omega_r$  upon that  $\omega_0$  this is actually nothing but  $\omega_r$  bar. So, this is  $2H d \omega_r$  bar  $d t$  is equal to  $T_m$  bar minus  $T_e$  bar; that means, all this quantities are in per unit right.

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Noting that  $T_{base} = \frac{VA_{base}}{\omega_{om}}$ , (111)

$$2H \frac{d\bar{\omega}_r}{dt} = \bar{T}_m - \bar{T}_e \dots (178) \rightarrow \text{X}$$

where

$$\bar{\omega}_r = \frac{\omega_m}{\omega_{om}} = \frac{\frac{\omega_r}{p_f}}{\frac{\omega_0}{p_f}} = \frac{\omega_r}{\omega_0}$$

where  $\omega_r$  is angular velocity of the rotor

So, and this is no need to read this is equation 178. So, we have seen  $\omega_r$  bar is equal to  $\omega_m$  upon  $\omega_0$  So, this we can write that  $\omega_r$  upon  $p_f$  divided by  $\omega_0$  upon  $p_f$  right this is basically your field poles  $p_f$  is field poles is equal to we can write  $\omega_r$  upon  $\omega_0$  right.

So, actually things are simple just to we are transforming it to per unit values.

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where  $\omega_r$  is angular velocity of the rotor in electrical rad/sec,  $\omega_0$  is its rated value and  $p_f$  is number of field poles.

If  $\delta$  is the angular position of the rotor in electrical radians with respect to a synchronously rotating reference and  $\delta_0$  is its value at  $t=0$ , then

$$\delta = (\omega_r - \omega_0)t + \delta_0 \dots (179) \rightarrow 199$$

So, where  $\omega$  is the angular velocity of the rotor in electrical radiant per second and  $\omega_0$  it is rotors value and  $p_f$  is the number of filed poles right. So, if  $\delta$  is the

angular position of the rotor in electrical radians with respect to a synchronously rotating reference and  $\delta_0$  is its initial value at  $t$  is equal to 0 then we can write  $\delta$  is equal to  $\omega_r$  minus  $\omega_0$  into  $t$  plus  $\delta_0$ . This is equation 179.

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electrical radians with respect to a synchronously rotating reference and  $\delta_0$  is its value at  $t=0$ , then

$$\delta = (\omega_r - \omega_0)t + \delta_0 \quad \dots (179) \rightarrow \text{X}$$

$$\therefore \frac{d\delta}{dt} = \omega_r - \omega_0 = \Delta\omega_r \quad \dots (180) \rightarrow \text{X}$$

and

$$\frac{d^2\delta}{dt^2} = \frac{d\omega_r}{dt} = \frac{d(\Delta\omega_r)}{dt}$$

$$\therefore \frac{d^2\delta}{dt^2} = \omega_0 \cdot \frac{d}{dt} \left( \frac{\omega_r}{\omega_0} \right) = \omega_0 \cdot \frac{d(\Delta\omega_r)}{dt} = \omega_0 \frac{d(\Delta\omega_r)}{dt}$$

So, this is no need, only this is equation 179 and this is equation 180. That means, if you take the derivative of it so,  $d\delta$  by  $dt$  will be  $\omega_r$  minus  $\omega_0$  and this is nothing but is equal to  $\Delta\omega_r$  right. If you take the double derivative of it and  $d^2\delta$  is equal to  $dt^2$  is equal to  $d\omega_r$  by  $dt$  is equal to  $dt$  of  $\Delta\omega_r$  right.

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then

$$\delta = (\omega_r - \omega_0)t + \delta_0 \quad \dots (179) \rightarrow 199$$

$$\therefore \frac{d\delta}{dt} = \omega_r - \omega_0 = \Delta\omega_p \quad \dots (180) \rightarrow 200$$

and

$$\frac{d^2\delta}{dt^2} = \frac{d\omega_r}{dt} = \frac{d(\Delta\omega_p)}{dt} = \frac{d\left(\omega_0 \left(\frac{\omega_r}{\omega_0}\right)\right)}{dt} = \omega_0 \frac{d\left(\frac{\omega_r}{\omega_0}\right)}{dt}$$

$$\therefore \frac{d^2\delta}{dt^2} = \omega_0 \cdot \frac{d\left(\frac{\omega_r}{\omega_0}\right)}{dt} = \omega_0 \frac{d\omega_r}{\omega_0 dt} = \omega_0 \frac{d(\Delta\omega_p)}{dt}$$

- - (181)  $\rightarrow 200$ .

Therefore, this  $\frac{d^2\delta}{dt^2}$  is equal to  $\omega_0$  right into your  $\frac{d^2\delta}{dt^2}$  of  $\omega_0$  mean you numerator and denominator you multiple by  $\omega_0$ . So, this equation you are writing  $\omega_0 \frac{d^2\delta}{dt^2}$  and  $\omega_0 \omega_r$  by  $\omega_0$  we are dividing it by  $\omega_0$  and here it is  $\omega_0$  is equal to we can write  $\omega_0$ . Then  $\frac{d\omega_r}{dt}$  of  $\omega_0 \omega_r$  bar, because this is if now per unit values right, is equal to we can write that  $\omega_0 \frac{d^2\delta}{dt^2}$  of  $\omega_0 \omega_r$  right. Because, earlier we have seen that you are here that  $\omega_r$  bar is equal to  $\omega_r$  upon  $\omega_0$  right.

And here it is your  $\delta \omega_r$   $\omega_r$  minus  $\omega_0$  is your  $\delta \omega_r$  right therefore, this one if you take that your  $\omega_0$  into  $\frac{d\omega_r}{dt}$  this is  $\omega_r$  bar right. So, is equal to we can write  $\omega_0 \frac{d^2\delta}{dt^2}$  of  $\delta \omega_r$ . I mean this one also you can I mean this is  $\delta \omega_r$ . So, if you take the derivative of this one it is  $\frac{d\omega_r}{dt}$  that is  $\frac{d^2\delta}{dt^2}$  of  $\delta \omega_r$ .

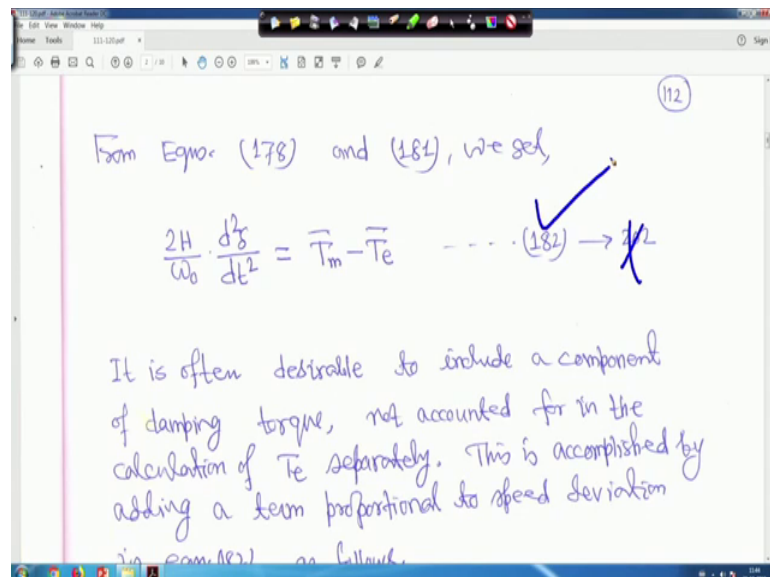
So, that means, we can make it the  $\frac{d\omega_r}{dt}$   $\omega_r$  by  $\frac{d^2\delta}{dt^2}$  is equal to  $\frac{d^2\delta}{dt^2}$  of  $\delta \omega_r$  bar I mean it is something like this it is something like this. Suppose you take this, this equation you take this equation right say both side both side you divided by your what you call that 0 right is  $\omega_0$   $\omega_0$  is equal to  $\frac{d^2\delta}{dt^2}$  of  $\delta \omega_r$  by  $\omega_0$  right.

So, this will become your  $\frac{d\omega_r}{dt}$   $\omega_r$  bar  $\frac{d^2\delta}{dt^2}$ , because  $\omega_r$  bar this one and  $\frac{d^2\delta}{dt^2}$  of  $\delta \omega_r$  upon  $\omega_0$  this is your base value. So, this will be per unit values,

so it will be  $\Delta \omega_r$  that is  $\Delta \omega_r$  bar d d t of  $\Delta \omega_r$  bar right. So, from this only from this only we are writing it that this is very this is very interesting from this we are writing  $\omega_0$  d d t of  $\omega_r$  bar is nothing but  $\omega_0$  d d t of  $\Delta \omega_r$  bar r bar right. So, this is equation your what you call this is equation 181 and this one this is nothing, this is for my I own reference because this is my class note. So, this is actually equation 181.

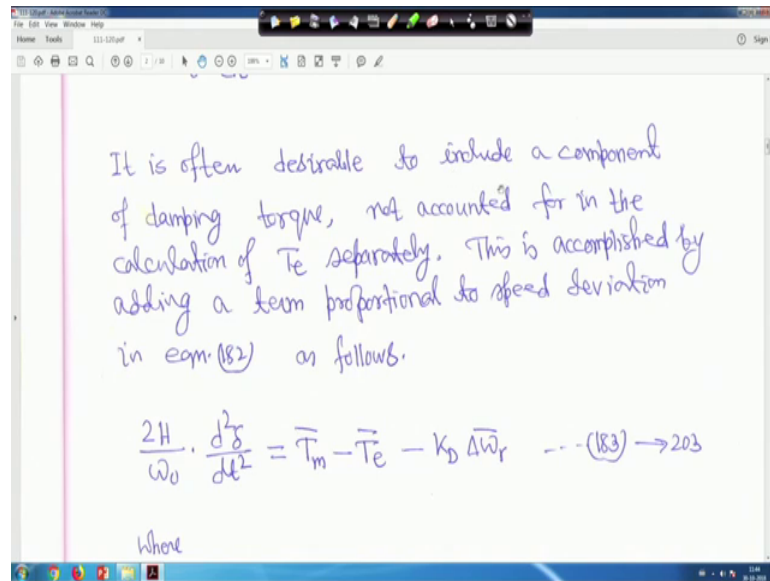
So, that means, from equation 178 and 181 you will get  $2H$  upon  $\omega_0$  d square delta upon d t square is equal to  $T_m$  bar minus  $T_e$  bar all are in per unit.

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This is equation 182 this is actually nothing only this one right.

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So, next it is often desirable to include a component of dampening torque right and not accounted for in the calculation of  $T_e$  separately. This is accomplished by adding a term proportion to speed the speed deviation in equation 182 as follows. Therefore, we can add one dampening term that is  $2H$  is upon  $\omega_0$   $d^2\delta$  upon  $dt^2$  is equal to  $\bar{T}_m$  minus  $\bar{T}_e$  minus  $K_D$  into  $\Delta\bar{\omega}_r$  this is equation 183. This one you should not see, this is my again and again I am telling this is my class note, so this is for my own reference right.

So,  $\bar{T}_m$  minus  $\bar{T}_e$  minus  $k_d$  into  $\Delta\bar{\omega}_r$  that is equation this dampening term is added right.

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$$\frac{2H}{\omega_0} \cdot \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - K_D \Delta\bar{\omega}_r \quad \dots (183) \rightarrow 203$$

Where

$$\Delta\bar{\omega}_r = \frac{\Delta\omega_r}{\omega_0} = \frac{1}{\omega_0} \cdot \frac{d\delta}{dt}$$

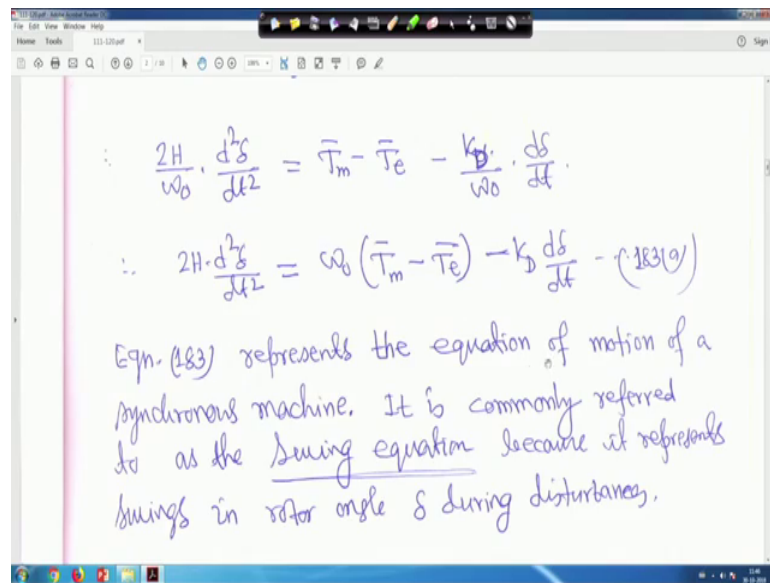
$$\therefore \frac{2H}{\omega_0} \cdot \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - \frac{K_D}{\omega_0} \cdot \frac{d\delta}{dt}$$

$$\therefore 2H \cdot \frac{d^2\delta}{dt^2} = \omega_0 (\bar{T}_m - \bar{T}_e) - K_D \frac{d\delta}{dt} \quad \dots (183a)$$

Where delta omega bar I told you earlier is equal to delta omega bar upon omega 0 right is equal to we can write 1 upon omega 0. And this is actually d delta by d t right delta omega r. Therefore, the 2H by omega 0 d square delta upon d t square is equal to T m bar minus T e bar, then these term minus KD term is there right and what we actually has been done that is delta omega r bar is equal to 1 upon omega 0 into d delta by d t this thing, this thing that your this omega bar is equal to this one has been your substituted here.

If your substitute here you will get minus KD upon omega 0 into d delta by d t right. Therefore this equation is coming it is here right. Therefore, 2H now both side you multiply by omega 0. So, it will be 2H d square delta upon d t square is equal to omega 0 in bracket T m bar minus T e bar minus KD into d delta by d t if you multiplying both side by omega 0 this omega 0 will be cancelled. So, minus KD upon a KD into delta by d t this is equation 183 a right.

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$$\therefore \frac{2H}{\omega_0} \cdot \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - \frac{k_D}{\omega_0} \cdot \frac{d\delta}{dt}$$

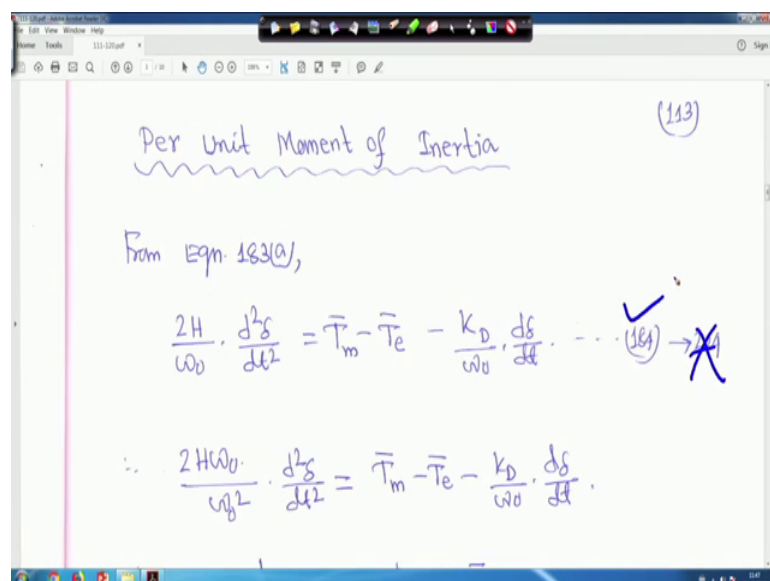
$$\therefore 2H \cdot \frac{d^2\delta}{dt^2} = \omega_0 (\bar{T}_m - \bar{T}_e) - k_D \frac{d\delta}{dt} \quad (183(a))$$

Eqn. (183) represents the equation of motion of a synchronous machine. It is commonly referred to as the swing equation because it represents swings in rotor angle  $\delta$  during disturbances.

So, equation 183 represents the equation of motion of a synchronous machine right; I mean this equation, this equation 183 right. It is commonly referred to as the swing equation, because it represents swing in rotor angle delta during disturbances right. So, it is commonly referred to as a swing equation, because it represents swings in rotor angle delta during very disturbances.

So, next is per unit moment of inertia.

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Per Unit Moment of Inertia (113)

From Eqn. 183(a),

$$\frac{2H}{\omega_0} \cdot \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - \frac{k_D}{\omega_0} \cdot \frac{d\delta}{dt} \dots (184) \rightarrow \times$$

$$\therefore \frac{2H\omega_0}{\omega_0^2} \cdot \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - \frac{k_D}{\omega_0} \cdot \frac{d\delta}{dt}$$



So, from equation 183 a; that means, these equation from this equation right  $2H$  upon  $\omega_0$  d square delta d t square  $T_m$  bar minus  $T_e$  bar minus  $k_D$  d upon  $\omega_0$  into d delta upon d t. This is equation 184, this one you should not see, this one you should not see right. This is this is equation 184.

So, therefore you multiply this equation numerator and denominator by  $2H \omega_0$ , so by  $\omega_0$ . So, numerator will be  $2H \omega_0$  and denominator will be  $\omega_0$  square into d square delta upon d t square is equal to  $T_m$  bar minus  $T_e$  bar minus  $k_D$  d upon  $\omega_0$  into d delta by d t right.

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$$\frac{2H}{\omega_0} \cdot \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - \frac{k_D}{\omega_0} \cdot \frac{d\delta}{dt} \quad \dots (184) \rightarrow 204$$

$$\therefore \left( \frac{2H\omega_0}{\omega_0^2} \right) \cdot \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - \frac{k_D}{\omega_0} \cdot \frac{d\delta}{dt}$$

$$t_{base} = \frac{1}{\omega_0} \quad \therefore \frac{1}{t_{base}} = \bar{\omega} = \frac{1}{\omega_0} = t_{base}^2$$

$$\therefore (2H\omega_0)^2 t_{base}^2 \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - k_D t_{base} \frac{d\delta}{dt}$$

Now we know that earlier we have seen  $t_{base}$  is nothing but  $1$  upon  $\omega_0$ . So, it is  $1$  upon  $\omega_0$  there is a time base right. Therefore,  $t$  by  $t_{base}$  equal to  $\bar{t}$  actually this is actually time it is actually  $t$  by  $t_{base}$  is equal to  $\bar{t}$  right time with if you take time in per unit and  $t_{base}$  is equal to  $1$  upon  $\omega_0$ .

If we do so that, that means, my  $t_{base}$  is  $1$ . That means, my  $1$  upon  $\omega_0$  square will be my  $t_{base}$  square, it is coming from here, it is coming from here right. That the  $1$  upon  $\omega_0$  square you can write  $2H \omega_0$  is, if this is  $2H \omega_0$  and one upon  $\omega_0$  square it is  $t_{base}$  square into d square delta d t square equal to  $T_m$  bar minus  $t T_e$  or  $T_e$  bar and  $1$  upon  $\omega_0$  is equal to  $t_{base}$ , because  $1$  upon  $\omega_0$  is equal to your  $t_{base}$ . So, we are multiplying minus  $K_D$  into  $t_{base}$ , because with the  $1$  upon  $\omega_0$  into d delta by d t this way we write right.

Next is and if we want to represent the timing per unit generally we represent in our all the quantities in per unit except time, but if we want to represent timing per unit. So what should be the technique?

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$$\therefore 2H\omega_0 \cdot t_{base}^2 \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - K_D t_{base} \frac{d\delta}{dt}$$

$$\therefore 2H\omega_0 \cdot \frac{d^2\delta}{d(\frac{t}{t_{base}})^2} = \bar{T}_m - \bar{T}_e - K_D \frac{d\delta}{d(\frac{t}{t_{base}})}$$

$$\therefore 2H\omega_0 \frac{d^2\delta}{d\bar{t}^2} = \bar{T}_m - \bar{T}_e - K_D \frac{d\delta}{d\bar{t}} \quad \dots (185)$$

Some authors refer to  $2H\omega_0$  as the perunit

Then this  $2H\omega_0$  is then this is  $d^2\delta/dt^2$ . So, we can write  $d^2\delta/dt^2$  upon  $t_{base}^2$  right because, this  $t_{base}^2$  has been your has a has been retained here  $t$  by  $t$  here what you call that one  $t$  is missing, that is your it is  $t_{base}$  right it is  $t_{base}$  right  $t_{base}^2$ . So,  $d^2\delta/dt^2$  upon  $d(t/t_{base})^2$ , because this one has been taken here that is by this thing or what we call that a  $t$  by  $t_{base}$  right. And similarly here also it is your  $-\dot{K}_D d\delta/dt$  upon  $t_{base}$  this has been written like this.

Therefore  $t/t_{base}$  is nothing but  $\bar{t}$  the timing per unit. Therefore, we can write  $2H\omega_0$  then  $d^2\delta/d\bar{t}^2$  is equal to  $\bar{T}_m - \bar{T}_e - K_D d\delta/d\bar{t}$ . That means, all these quantities for this equation 185 within per unit including time, if we want to represent time in per unit generally in stability studies only time we generally represent in second, but other quantities we here what we call represented per unit values right.

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$$\therefore 2H\omega_0 \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - K_D \frac{d\delta}{dt} \quad \dots (185)$$

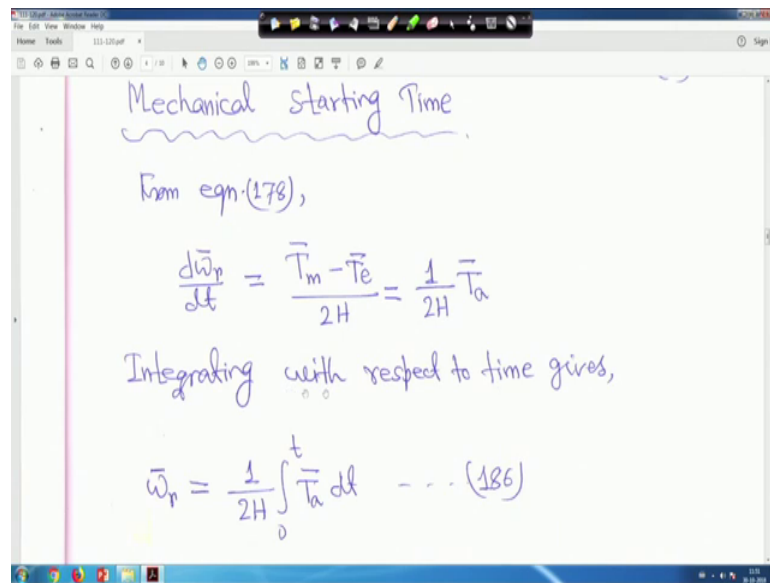
Some authors refer to  $2H\omega_0$  as the per unit moment of inertia  $\bar{J} = 2H\omega_0$ .

$$\bar{J} \frac{d^2\delta}{dt^2} = \bar{T}_m - \bar{T}_e - K_D \frac{d\delta}{dt}$$

So, now some authors have written I mean many researches or authors they refer to  $2H\omega_0$  as the per unit moment of inertia that is  $\bar{J}$  is equal to  $2H\omega_0$ , because H you have seen from your stability studies that unit or dimension of H in second inertia is in second and  $\omega_0$  is radian per second.

So finally,  $2H\omega_0$  will be dimension less quantity that is why some authors they refer this one as the  $2H\omega_0$  as the per unit moment of inertia. That is why you are putting  $\bar{J}$ . That means, everything if you see that everything will be per unit. That this equation if I write like this that  $\bar{J}$  then  $d^2\delta/dt^2$  is equal to  $\bar{T}_m$  minus your  $\bar{T}_e$  minus  $K_D$  then your  $d\delta/dt$  right. So, everything is in per unit right all these quantities are in per unit.

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Mechanical Starting Time

From eqn. (178),

$$\frac{d\bar{\omega}_r}{dt} = \frac{\bar{T}_m - \bar{T}_e}{2H} = \frac{1}{2H} \bar{T}_a$$

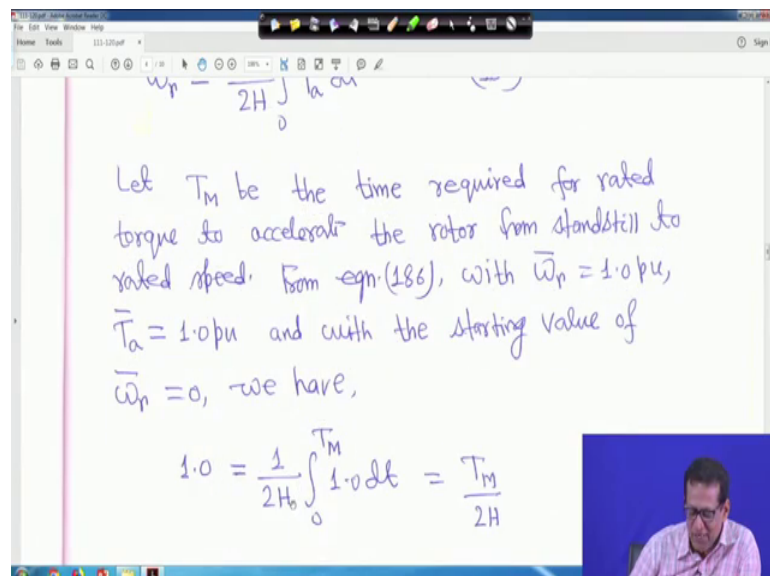
Integrating with respect to time gives,

$$\bar{\omega}_r = \frac{1}{2H} \int_0^t \bar{T}_a dt \quad \dots (186)$$

So, now next is that a mechanical starting time. Now from equation 78, 178 rather from equation 178, we can write the d d t of omega r bar is equal to T m bar minus T e bar upon 2H right is equal to we can write 1 upon 2H t a bar that is the accelerating power t a is equal to T m minus T e you know. So, t a bar will be T m bar minus T e bar, so 1 upon 2H t a bar right. Now if you integrate with respect to time then omega r will give one upon 2H 0 to t t a bar d t this is equation 186 right.

Now, this is some of calculations that your mechanical starting time.

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$\bar{\omega}_r = \frac{1}{2H} \int_0^t \bar{T}_a dt$

Let  $T_m$  be the time required for rated torque to accelerate the rotor from standstill to rated speed. From eqn. (186), with  $\bar{\omega}_r = 1.0$  pu,  $\bar{T}_a = 1.0$  pu and with the starting value of  $\bar{\omega}_r = 0$ , we have,

$$1.0 = \frac{1}{2H} \int_0^{T_m} 1.0 dt = \frac{T_m}{2H}$$

Therefore, now let  $T_m$  be the time required for rated or to accelerate the rotor from standstill to rotor speed right. I mean if you assume the  $T_m$  be the time required for rated torque to accelerate the rotor from standstill to rated speed. Now from equation 186 this one with  $\omega_r$  bar is equal to say 1 per unit and  $t_a$  bar is equal to say 1.0 per unit and with the starting value of  $\omega_r$  bar is equal to 0, say initial value that is starting value is 0.

Then, you can see that your what you call 1 is equal to because  $\omega_r$  we have taken 1, 1 upon  $2H$   $0$   $2$   $2$   $m$  here also  $t_a$  bar you have taken 1 right and to  $d t$ , so it is  $T_m$  upon  $2H$  right.

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The image shows a whiteboard with handwritten text and equations. The text reads: " $T_a = 1.0 \text{ pu}$  and with the starting value of  $\omega_r = 0$ , we have,". Below this, the equation  $1.0 = \frac{1}{2H} \int_0^{T_m} 1.0 dt = \frac{T_m}{2H}$  is written. This is followed by the conclusion  $\therefore T_m = 2H \text{ Sec.}$ . At the bottom, it states " $T_m$  is called the mechanical starting time."

That means, we can write  $T_m$  is equal to  $2H$  second. So,  $T_m$  is called the mechanical starting time. It is some rough, it is some approximate calculation. So,  $T_m$  is called the mechanical starting time. So, it is  $2H$  second because dimension are  $h$  is in second.

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Calculation of Inertia Constant

$$H = \frac{\text{Stored energy at rated speed in MW-sec}}{\text{MVA rating}}$$

Stored energy = Kinetic energy

$$= \frac{1}{2} J \omega_m^2 \text{ Watt-sec}$$
$$= \frac{1}{2} J \omega_m^2 \times 10^{-6} \text{ MW-sec}$$

Now, calculation of inertia constant; so now we know from our power system studies, power system analysis transient stability studies are in machine that H is equal to store energy at rated speed in megawatt second divided by MVA rating right. So, its unit is second right. Therefore, stored energy is nothing but is equal to kinetic energy. So, we can write kinetic energy is equal to stored energy half J omega 0 m square that is watt second.

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$$= \frac{1}{2} J \omega_m^2 \times 10^{-6} \text{ MW-sec}$$

Where

$J = \text{moment of inertia in kg-m}^2$

$\omega_m = \text{rated speed in mechanical rad/sec}$

$$= \frac{2\pi \text{ RPM}}{60}$$

Therefore,

Where the energy is joule and we know joule per second is equal to watt. So, joule is equal to watt second. So, we write in watt second rather than joule right. So, we can write, now we if you convert watt into megawatt then it will be half J omega 0 m square into 10 to the power minus 6 megawatt second right because 10 to the power 6 watt is equal to 1 megawatt right.

So, that is why your half J omega 0 m square in to 10 to the power minus 6 megawatt second. Now J is equal to your moment of inertia in kg meter square, this we know. Omega 0 a means rated speed in mechanical radian per second right, is equal to 2 pi into revolution per minute RPM by 60. That is omega 0 m right, in because we are giving in radian per second.

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$$= \frac{2\pi \text{ RPM}}{60}$$

Therefore,

$$H = \frac{1}{2} \frac{J \omega_m^2 \times 10^{-6}}{\text{MVA rating}}$$

$$\therefore H = \frac{1}{2} \frac{J (2\pi \text{ RPM}/60)^2 \times 10^{-6}}{\text{MVA rating}}$$

$$\therefore H = 5.48 \times 10^{-9} \frac{J (\text{RPM})^2}{\text{MVA rating}}$$

Now therefore, this H will be is equal to half J omega 0 m square in 10 to the power minus 6 divided by MVA rating right because, this is that equation, this is the equation and divided by MVVA rating. Now this is equal to H is equal to half J omega 0 m is equal to 2 pi the revolution per minute, I write RPM divided by 60 square into 10 to the power minus 6 divided by MVA rating. If you simply this one it comes actually 5.48 into 10 to the power minus 9 J RPM square divided by MVA rating; that is equation 187.

Now, this is actually value of H, this is from for the point of view of solving numerical. This you have to keep it in your mind, I mean keep it in your memory right.

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Typical Values of H

Thermal Unit

3600 rpm or 3000 rpm (2-pole)  $2.5 \leq H \leq 6.0$

1800 rpm or 1500 rpm (4-pole)  $4.0 \leq H \leq 10.0$

Hydraulic Unit

$2.0 \leq H \leq 4.0$

So, typical value of H; so generally for a machine that 3 6 3600 RPM or 3000 RPM right basically 2 pole machine for thermal power plant thermal unit is generally lies in between 2.5 and 6.0 second right. And if it is 1800 RPM or 1500 RPM that is 4 pole machines. So, it generally lies in between 4 and 10 second right. And similarly, for hydro unit hydraulic unit H is generally it by typical values lies in between 2 and 4 second right.

So, this is actually here what you call different values of H range or values of H for thermal or hydro power plant.



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Hydraulic Unit

$$2.0 \leq H \leq 4.0$$

Representation in System Studies

The state-space form requires the component models to be expressed as a set of first order differential equations. The swing equation (183) expressed as two first order differential equations becomes

Now, represents as represent as in system studies right. The now the state space form requires that your what you call the component models to be expressed as a set of faster order differential equation because, slowly and slowly we have to go for the here, what you call development of your first order. Now here, what you call block diagram representation of the synchronous machine right.

So, the state space form requires your component models to be expressed as a set of first order differential equations.

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expressed as two first order differential equations, becomes

$$\frac{d(\Delta\omega_p)}{dt} = \frac{1}{2H} (\bar{T}_m - \bar{T}_e - K_D \Delta\omega_p) \quad (188)$$

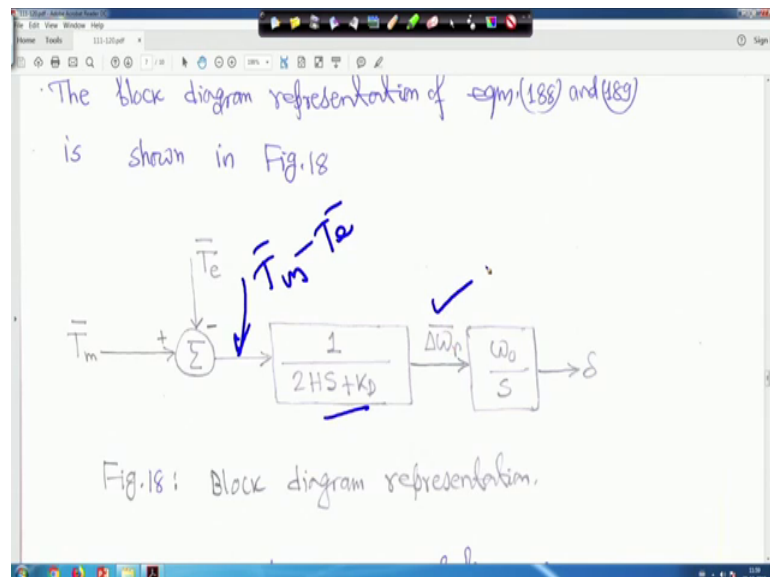
$$\frac{d\delta}{dt} = \omega_0 \Delta\omega_p \quad (189)$$

In the above equations, time in 't' is in seconds, rotor angle 'δ' is in electrical radians, and  $\omega_0$  is equal to  $2\pi f_s$ .

The swing equation that is 183 expressed as 2 first order differential equations it becomes actually  $\frac{d}{dt} \Delta \omega$  is equal to  $\frac{1}{2H} (T_m - T_e - K_D \Delta \omega)$ . This is equation your 188 right. So, this is 100, this is nothing, this is also nothing and  $\frac{d}{dt} \Delta \omega$  is equal to  $\omega_0 \Delta \delta$ . This we have seen this right.

Now, in the above equation time  $t$  is in second, because here we are not using  $\bar{t}$ , it is time in second at rotor angle  $\delta$  is in electrical radiant and  $\omega_0$  is equal to  $2\pi f$  right.

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So, now the block diagram representation of equation 188. That means, this 2 equation and 189 is shown in figure 18 right.

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$$\frac{d(\Delta\omega_r)}{dt} = \frac{1}{2H} (\bar{T}_m - \bar{T}_e - K_D \Delta\omega_r) - (188)$$

$$\frac{d\delta}{dt} = \omega_0 \Delta\omega_r (2HS + K_D) \Delta\omega_r = \frac{\bar{T}_m - \bar{T}_e}{2HS + K_D}$$

In the above equations, time in 't' is in seconds, rotor angle 'δ' is in electrical radians and  $\omega_0$  is equal to  $2\pi f$ .

$$S \Delta\omega_r = \frac{1}{2H} (\bar{T}_m - \bar{T}_e - K_D \Delta\omega_r)$$

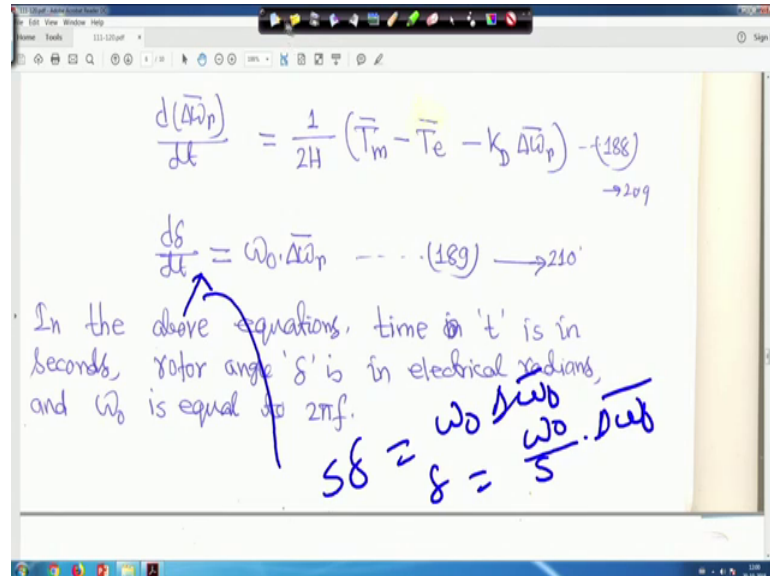
Now, question is that assuming all initial conditions are 0 if you take the Laplace transform of this one right. So it will become S into delta your delta omega bar S S will not put again and again right understandable. So, it will be s into delta omega r is equal to one upon 2H it will be T m T m bar minus T e bar minus KD delta omega r bar right. I mean if you if you look into this that if you take the Laplace transform on both side, so again and again I am not making this quantities function of S, understandable only S I will put. So, it is S right then, delta omega bar is equal to 1 upon 2H right. Then T m bar minus T e bar minus KD then your delta omega r bar right this way we can write.

Now, if you go for cross multiplication and simplify then where to write I am writing here writing here just see that then 2H S, we go for cross multiplication and after cross multiplication d KD omega r term to left hand side. So, it will be KD right into your delta omega r bar because here delta omega bar, here also delta omega bar is equal to your T m bar minus T e bar right. So, 2H S plus KD term will be there. Therefore, delta omega bar will be your T m minus T e divided by 2H S plus KD right.

So, that is why this block diagram, this block diagram if we look into that T m bar is given input here and T e bar. So, this one this output is T m bar minus T e bar into and delta omega r will be your what you call your T m bar minus T e bar divided by 2H S plus KD, that is you have seen and this output is delta omega r bar. Now again if you see this equation that your this equation if you take your what you call that your Laplace

transform a again and again not putting the function of this thing just hold on, just a this thing than this one, if you take S delta is equal to omega 0 delta omega r bar right.

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This is there, therefore delta will be is equal to omega 0 by a S into delta omega r bar right.

So, that is why here we are writing the delta is equal to omega 0 by S into delta of this is the first step, slowly and slowly it will grow right. I mean just step by step it will grow, later you will see it will make a very big model just hold on. Slow and slowly we are it will be growing that step by step right.

So, this is actually then we can write T m bar my then this is T m bar and this is minus one T e bar and this block diagram you put one upon 2H S plus KD. This output is delta omega bar and omega 0 by S into and this output is delta right. So, up to this is no problem at all it is very simple thing right.

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Fig. 18: Block diagram representation.

Synchronous Machine Representation in Stability Studies

For stability analysis of large systems, it is necessary to neglect the following from eqns. (118) & (119) for stator voltage:

- The transformer voltage terms,  $p\psi_d$  and  $p\psi_q$

The screenshot shows a whiteboard with handwritten text in blue ink. At the top, it says 'Fig. 18: Block diagram representation.' Below that is the title 'Synchronous Machine Representation in Stability Studies' underlined. The main text discusses neglecting terms from equations 118 and 119 for stator voltage. A small video inset of a man is visible in the bottom right corner of the whiteboard area.

This is my figure 18; that is block diagram representation. Now synchronous machine representation in stability studies we have to make few assumptions. Now for stability analysis of large system it is necessary to neglect the following terms from equation 118 and 119 for stator voltage. So, whenever you will whenever you will listen to this course right this as all this notes will be provided to you. So, keep everything ready in front of you because if I do not want to go back again a 118 and 119 for examples many such things will come, so you will keep those things in front of you and just see this right; so that from equation 118 and 19 first stator voltage right.

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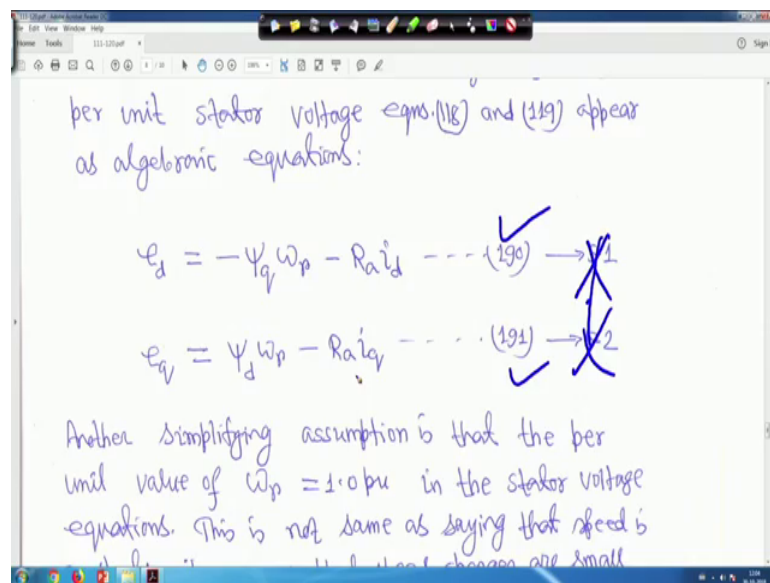
- The transformer voltage terms,  $p\psi_d$  and  $p\psi_q$ .
- The effect of speed variations.

The  $p\psi_d$  and  $p\psi_q$  terms represent the stator transients. With these terms neglected, the stator quantities contain only fundamental frequency components and the stator voltage equations appear as algebraic equations.

The screenshot shows a whiteboard with handwritten text in blue ink. It lists two bullet points: 'The transformer voltage terms,  $p\psi_d$  and  $p\psi_q$ .' and 'The effect of speed variations.' Below the list, it explains that these terms represent stator transients and that neglecting them results in algebraic equations for stator voltage.

It is necessary to neglect the following terms the first is that, transformer voltage terms  $p \psi_d$  that is  $p \psi_d \dot{\phantom{x}}$  and  $p \psi_q$  that is  $p \psi_q \dot{\phantom{x}}$ . This will be dropped right because we are and similarly the effect of speed variation, this also will be dropped right. If it is so the  $p \psi_d$  and  $p \psi_q$  terms represent the stator transients right. With this terms neglected the stator quantities contained only fundamental frequency components and the stator voltage equations appear as algebraic equation right.

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That means with the stator transients neglected the per unit stator voltage equation 118 and 119 appear as algebraic equation. So,  $p \psi_d$  and  $p \psi_q$  terms will be dropped in equation 118 and 119. I actually what I have done is this I did in big this thing, this equation number will increase right so such to maintain the continuity.

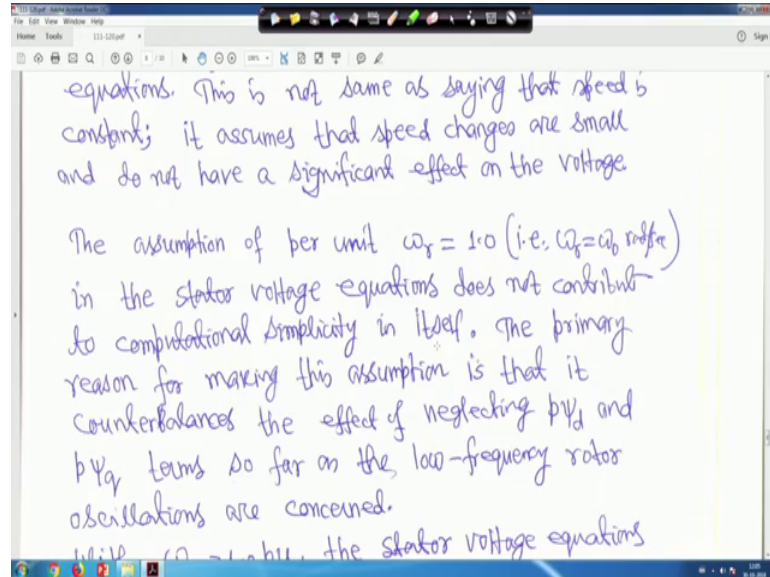
So, I perhaps a I feel that it will be very helpful for you people because when I have taught this course here they are student also find this things are quite useful right. So, that is why that equation number will continue and it will cross you can have so many equation such that there will be continuity a link right. That is why this equation number, so there nothing to be surprised looking at equation 190, 191 or 200 or more right, but it will help you a lot it because, continuity will be there and you can link 1 after another when we will have this nodes right.

So, that is why your, that means, this equation here what you call this equation, that your 118 and 119 right, so this is no need to see, this is no need to see. So, it is because of 190

and 191 right. Therefore,  $e_d$  will be minus  $\psi_q \omega_r$  and minus  $R_a i_d$  and  $e_q$  will be  $e_q$  will be  $\psi_d \omega_r$  minus  $R_a i_q$  after dropping  $p \psi_d$  and  $p \psi_q$  terms right.

So, another simplified assumption is we have to make many assumptions.

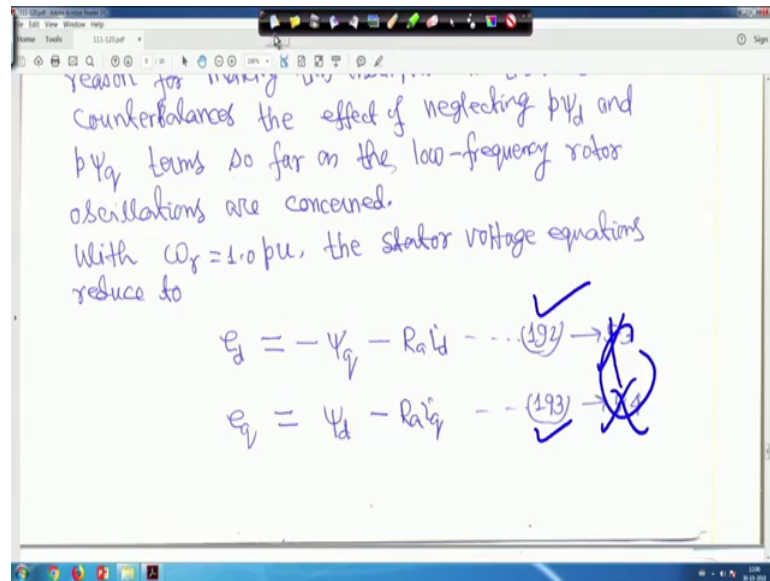
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Another simplifying assumption is that the per unit value of  $\omega_r$  is taken 1.0 per unit in the stator voltage equation. This is not same as saying that the speed is constant. It assume that speed changes are small and do not have a significant effect on the voltage right.

So, this is this sentence is very important that in the stator voltage equations this is not same as saying that speed is constant rather it assume that speed changes are small and do not have a significant effect on the voltage right. So, the assumptions per unit  $\omega_r$  is equal to your 1. That is  $\omega_r$  is equal to  $\omega_0$  is equal to a this thing  $\omega_0$  radian per second in the stator voltage equation does not contribute to computational simplicity in itself.

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The primary reason for making this assumption is that it counter balances the effect of neglecting  $p\psi_d$  and  $p\psi_q$  terms. So, far as the low frequency rotor oscillations are concerned. We will see those things in later right much later right.

With  $\omega_r$  is equal to 1.0 per unit the stator voltage equations reduce to that  $e_d$  will now here in this equation if  $\omega_r$  is equal to 1 per unit then this here what you call then  $e_d$  will be is equal to this is not required right. So, this for my own reference, so  $e_d$  is equal to minus  $\psi_q$  minus  $R_a i_d$ , this is equation 192 and  $e_q$  will be  $\psi_d$  minus  $R_a i_q$ , this is equation 193 right. So that means we are simplifying 1 after another right. So now we will.

Thank you very much. We will be back again.